

**Amendments to the Specification:**

Please replace the paragraph beginning at page 3, line 16, with the following rewritten paragraph:

In accordance with yet another aspect of the invention, the segmented labyrinth seal has a windback configuration formed around a rotatable shaft for preventing leakage of oil from a bearing housing. The seal includes two half-circle shaped segments having first and second ends. When assembled, the two half-circle shaped segments form a cylindrical shape, and the first ends abut one another and the second ends abut one another. The seal also includes an exterior cylindrical surface and an interior cylindrical surface, and the exterior cylindrical surface and the interior cylindrical surface extend between a first face and a second face. A thread pattern is provided on the interior cylindrical surface, and the thread pattern is selectively configured in a right-hand direction when the rotatable shaft is rotating clockwise when looking down the rotatable shaft toward the bearing housing and in a left-hand direction when the rotatable shaft is rotating counter-clockwise when looking down the rotatable shaft toward the bearing housing. First and second split-line pins are respectively located on the first and second ends of one of the two half-circle shaped segments, and first and second holes are respectively provided on the first and second ends of the other of the two half-circle shaped segments. The first hole receives the first split-line pin and the second hole hold receives the second split-line pin when the segmented labyrinth seal is assembled. The position of the first hole and the first split-line pin is staggered in relation to the second hole and the second split-line pin depending on the selective configuration of the thread pattern in the right-hand direction and the left-hand direction. The thread pattern is formed by a plurality of profiled teeth having first sides, second sides, and connecting sides extending between the first sides and the second sides. The plurality of profiled teeth have a vertical tooth height, and leading edges are formed where the first sides join the connecting sides and trailing edges are formed where the second sides join the connecting sides. The first sides and the second sides respectively form first and second angles that

are oblique with respect to the interior cylindrical surface, the first angle always being greater than the second angle. A pressure drop is taken over the plurality of profiled teeth, and is divided into various intermediate pressures between adjacent teeth of the plurality of profiled teeth. A channel which traces the thread pattern is wound in a direction opposite to the rotational direction of the rotatable shaft. The channel is adapted for capturing the oil from the bearing housing, and returning the oil to the bearing housing without the need for axial drain holes. The vertical tooth height of the plurality of profiled teeth is chosen to allow for a primary flow of said oil directed to the bearing housing in the channel, and to prevent secondary flow of the oil in an opposite direction to the primary flow in the channel.

Please replace the paragraph beginning at page 6, line 6, with the following rewritten paragraph:

The segmented labyrinth seal 10 will be formed around a rotatable shaft (not shown), and, as discussed above, will be encapsulated in the bearing housing. To facilitate encapsulation of the labyrinth seal 10, and eventual replacement of the first segment 11 and second segment 12, the segmented labyrinth seal 10 has DIN Standard dimensions, and the exterior surface 19 is consequently provided with a profile 25. The profile 25 extends around the segmented labyrinth seal 10, and is provided with a groove 26. The groove 26 is adapted to accommodate a hoop-shaped clamping spring (not shown), and fit within the above-referenced bearing housing. The clamping spring 27 is used to hold the first segment 11 and second segment 12 together around the rotatable shaft.